

Sliding Mode Control for Uncertain Thermal SOFC Models with Physical Actuator Constraints

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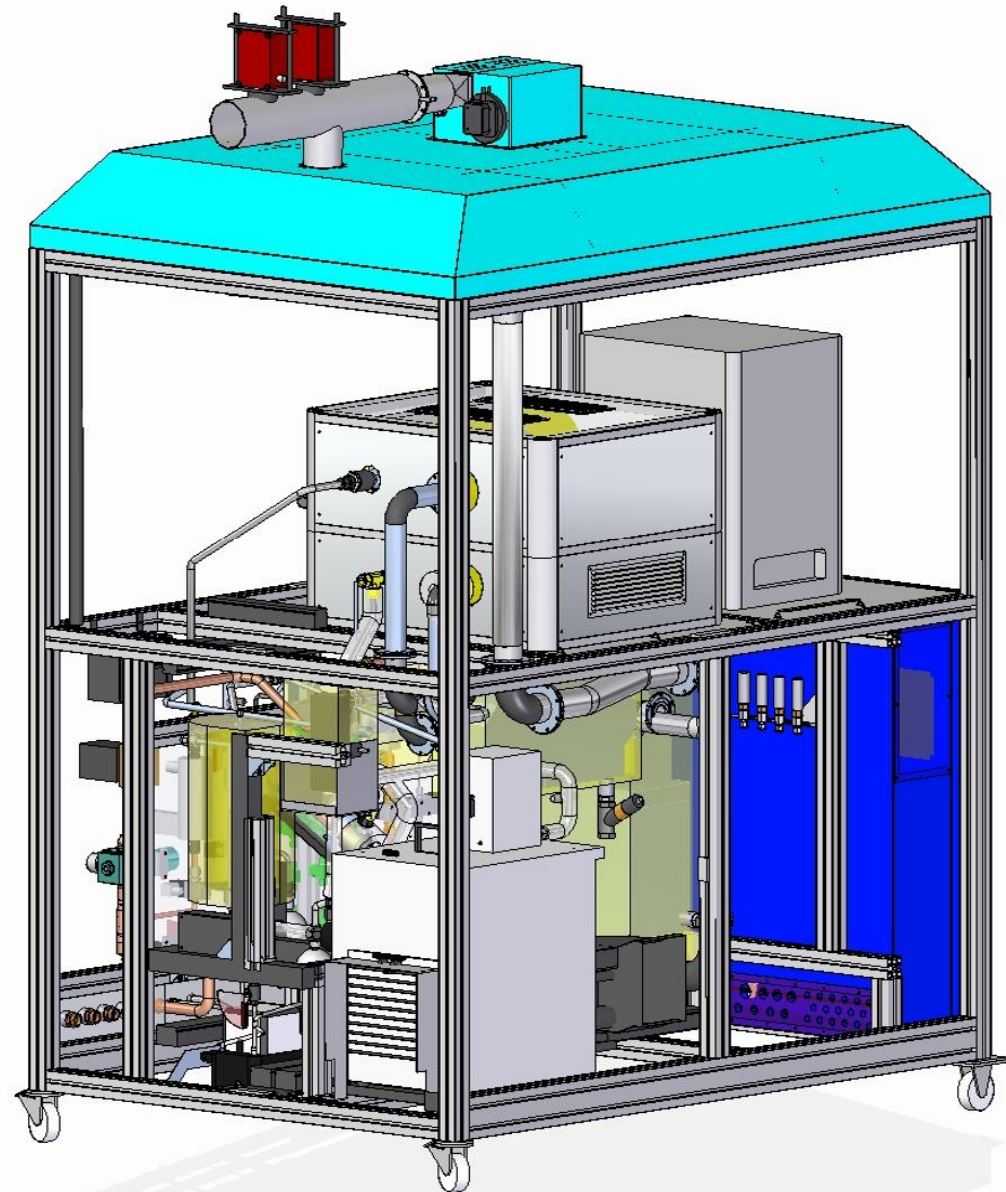
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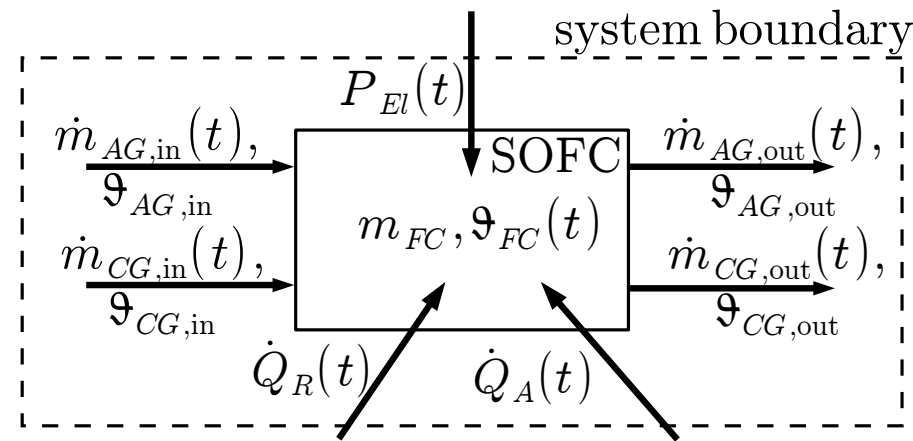
Working principles

- Fluid supply (fuel gas, air)
- Independent preheaters for fuel gas and air
- Stack module containing fuel cells in electric series connection
- Variable electric load as a disturbance



Modeling Approach

Energy balance of the SOFC stack module



- Control-oriented modeling of a SOFC stack module for the derivation of control and observer strategies
- Integral balancing of an instationary energy conversion process in the whole stack module as well as in individual finite volume elements
- Impact of the variation of the internal energy on the local temperature distribution in the stack module



Modeling Approach

- Relation between the variation of the internal energy and the stack temperature for constant material parameters c_{FC} and m_{FC}

$$\frac{dE_{FC}(t)}{dt} = c_{FC} \cdot m_{FC} \cdot \frac{d\vartheta_{FC}(t)}{dt}$$

- Modeling of the effects on the internal energy

$$\begin{aligned} \frac{dE_{FC}(t)}{dt} = & C_{AG}(\vartheta_{FC}, t) (\vartheta_{AG,in}(t) - \vartheta_{FC}(t)) \\ & + C_{CG}(\vartheta_{FC}, t) (\vartheta_{CG,in}(t) - \vartheta_{FC}(t)) \\ & + \dot{Q}_R(t) + P_{El}(t) + \dot{Q}_A(t) \end{aligned}$$

- Reaction heat flow of the hydrogen oxidation reaction

$$\dot{Q}_R = \frac{\Delta_R H(\vartheta_{FC}) \cdot \dot{m}_{H_2}(t)}{M_{H_2}}$$

Modeling Approach

- Relation between the variation of the internal energy and the stack temperature for constant material parameters c_{FC} and m_{FC}

$$c_{FC} \cdot m_{FC} \cdot \frac{d\vartheta_{FC}(t)}{dt} = C_{AG}(\vartheta_{FC}, t) (\vartheta_{AG,in}(t) - \vartheta_{FC}(t)) \\ + C_{CG}(\vartheta_{FC}, t) (\vartheta_{CG,in}(t) - \vartheta_{FC}(t)) \\ + \dot{Q}_R(t) + P_{El}(t) + \dot{Q}_A(t)$$

- Heat transfer including linearized heat radiation to ambient media

$$\dot{Q}_A = \frac{1}{R_A} (\vartheta_A - \vartheta_{FC})$$

- Ohmic loss effects in the stack material

$$P_{El}(t) = R_{El} I^2(t)$$

Modeling Approach

- Relation between the variation of the internal energy and the stack temperature for constant material parameters c_{FC} and m_{FC}

$$c_{FC} \cdot m_{FC} \cdot \frac{d\vartheta_{FC}(t)}{dt} = C_{AG}(\vartheta_{FC}, t) (\vartheta_{AG,in}(t) - \vartheta_{FC}(t)) \\ + C_{CG}(\vartheta_{FC}, t) (\vartheta_{CG,in}(t) - \vartheta_{FC}(t)) \\ + \dot{Q}_R(t) + P_{El}(t) + \dot{Q}_A(t)$$

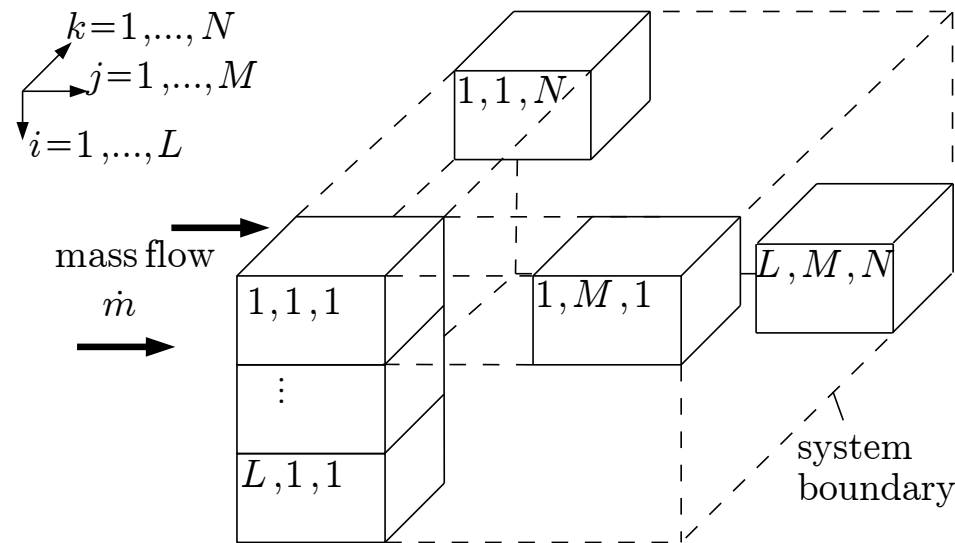
- Anode gas: Heat capacity approximated by 2nd-order polynomials for c_χ with $\chi \in \{H_2, N_2, H_2O\}$

$$C_{AG}(\vartheta_{FC}, t) = c_{H_2}(\vartheta_{FC}) \dot{m}_{H_2}(t) \\ + c_{N_2}(\vartheta_{FC}) \dot{m}_{N_2}(t) + c_{H_2O}(\vartheta_{FC}) \dot{m}_{H_2O}(t)$$

- Cathode gas: Heat capacity approximated with 2nd-order polynomials for c_{CG}

$$C_{CG}(\vartheta_{FC}, t) = c_{CG}(\vartheta_{FC}) \cdot \dot{m}_{CG}(t)$$

Semi-Discretization: The Finite Volume Method



- Semi-discretization into $n_x = L \cdot M \cdot N$ finite volume elements to describe the internal temperature distributions
- Local energy balances lead to a set of n_x coupled ODEs represented by a state vector $x^T = [\vartheta_{1,1,1}, \dots, \vartheta_{L,M,N}] \in \mathbb{R}^{n_x}$
- System boundary includes the thermal stack insulation



Semi-Discretization: The Finite Volume Method

- ODE for the local temperature distribution in a SOFC stack module

$$\begin{aligned}
 c_{i,j,k} \cdot m_{i,j,k} \cdot \dot{\vartheta}_{i,j,k}(t) = & C_{AG,i,j,k}(\vartheta_{i,j,k}, t) \left(\vartheta_{i,j-1,k}(t) - \vartheta_{i,j,k}(t) \right) \\
 & + C_{CG,i,j,k}(\vartheta_{i,j,k}, t) \left(\vartheta_{i,j-1,k}(t) - \vartheta_{i,j,k}(t) \right) \\
 & + \dot{Q}_{\eta,i,j,k}(t) + \dot{Q}_{R,i,j,k}(t) + P_{El,i,j,k}(t)
 \end{aligned}$$

- Modeling of local temperature-dependent and time-varying influence factors

Heat flow:
$$\dot{Q}_{\eta,i,j,k}(t) = \sum_{\eta \in \mathcal{N}} \frac{1}{R_{th,\eta}^{i,j,k}} (\vartheta_{\eta}(t) - \vartheta_{i,j,k}(t))$$

Reaction heat flow:
$$\dot{Q}_{R,i,j,k}(t) = \frac{\Delta_R H_{i,j,k}(\vartheta_{i,j,k}) \cdot \dot{m}_{H_2,i,j,k}(t)}{M_{H_2}}$$

Ohmic losses:
$$P_{El,i,j,k}(t) = R_{El,i,j,k} I_{i,j,k}^2(t)$$

Semi-Discretization: The Finite Volume Method

Case 1: Semi-discretization into a single finite volume element leads to the global energy balance described before

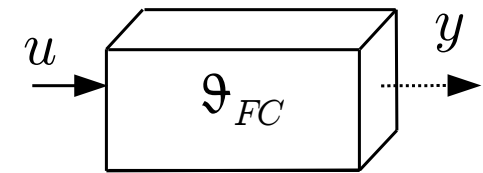
- State variable x and output variable y

$$x(t) = \vartheta_{FC}(t)$$

$$y(t) = h(x) = \vartheta_{FC}(t)$$

- Nonlinear ordinary differential equation

$$\dot{\vartheta}_{FC} = \Phi(\vartheta_{FC}(t), u(t))$$



Case 2: Semi-discretization into three finite volume elements oriented in the direction of mass flow

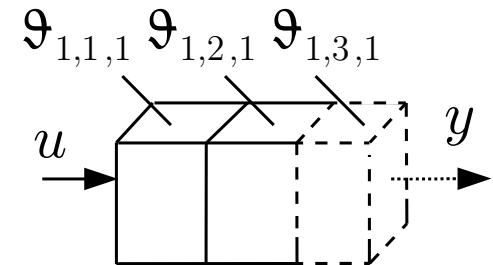
- State vector x and output variable y

$$x(t) = [\vartheta_{111}(t), \vartheta_{121}(t), \vartheta_{131}(t)]^T$$

$$y(t) = h(x) = \vartheta_{131}(t)$$

- Set of coupled nonlinear ordinary differential equations

$$\dot{x}(t) = \Phi(x(t), u(t))$$



State Equations — Simplification for Control Synthesis

- Input-affine description of the nonlinear thermal subsystem

$$\dot{x}(t) = f(x(t)) + g(x(t)) \cdot u(t), \quad x \in \mathbb{R}^{n_x}$$

$$y(t) = h(x(t)), \quad y \in \mathbb{R}^{n_y}$$

$$u(t) = \dot{m}_{CG} \cdot \Delta\vartheta(t)$$

- Realization of the temperature difference $\Delta\vartheta$ in the control input by an underlying controller for the preheating devices

$$\Delta\vartheta(t) := \begin{cases} \vartheta_{CG}(t) - \vartheta_{FC}(t) & \text{for } x(t) = \vartheta_{FC}(t) \\ \vartheta_{CG}(t) - \vartheta_{111}(t) & \text{for } x(t) = [\vartheta_{111}(t), \vartheta_{121}(t), \vartheta_{131}(t)]^T \end{cases}$$

- Exact input-output linearization with relative degree δ (Computation of the Lie-Derivatives of y)

$$\frac{d^i y}{dt^i} = L_f^i h(x) = L_f \left(L_f^{i-1} h(x) \right), \quad i = 0, \dots, \delta - 1$$

- Relative degree δ determines the smallest order that explicitly depends on the input u

Modeling Approach — Transformation of the State-Space

- Nonlinear transformation of the state equations with the relative degree $\delta = n_x$ according to

$$z^T = [z_1 \ z_2 \ z_3] = [y \ \dot{y} \ \ddot{y}] = [h(x) \ L_f h(x) \ L_f^2 h(x)]$$

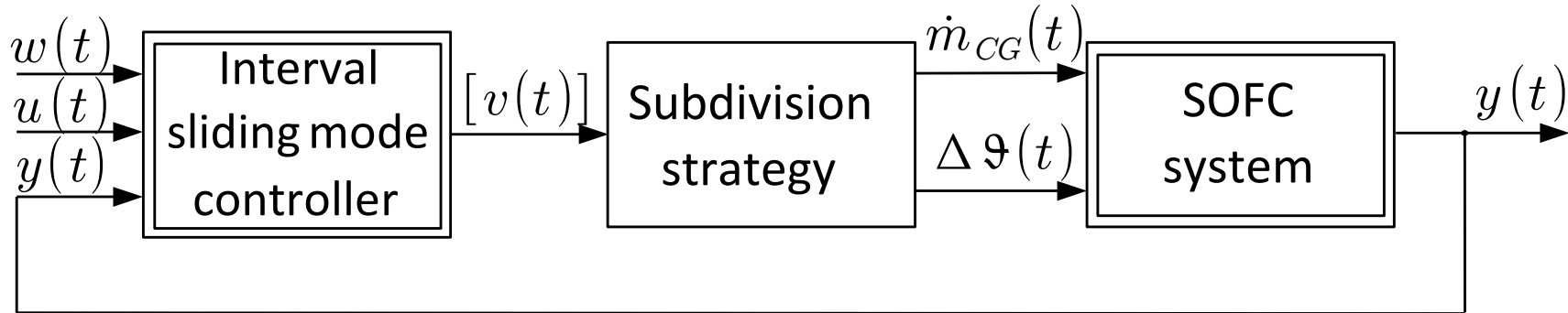
- Nonlinear controller normal form (NCNF)

$$\dot{z} = \begin{bmatrix} \dot{z}_1 \\ \dot{z}_2 \\ \dot{z}_3 \end{bmatrix} = \begin{bmatrix} L_f h(x) \\ L_f^2 h(x) \\ L_f^3 h(x) \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ L_g L_f^2 h(x) \end{bmatrix} u$$

- Feedback linearizing control law for sufficiently small variations of the mass flow used for the heat-up phase of the SOFC

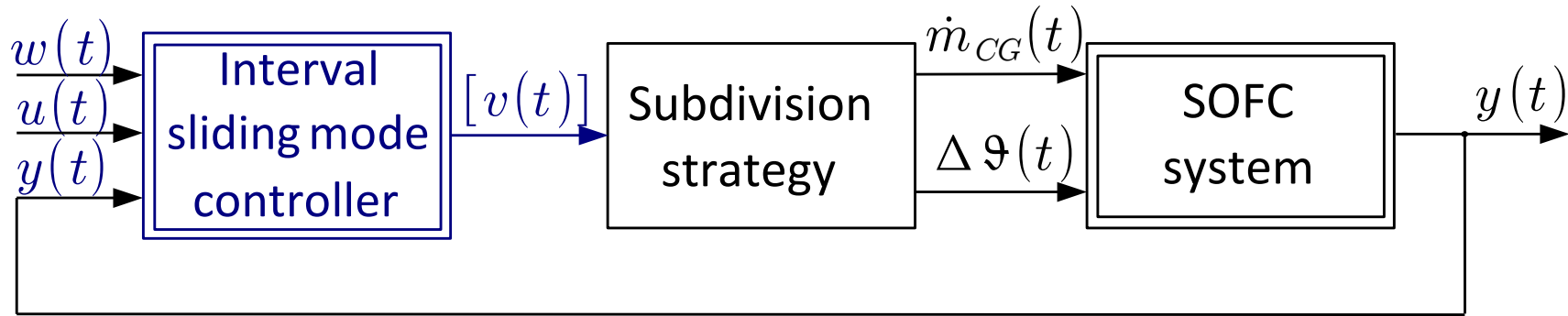
$$u := \frac{-L_f^3 h(x) - \alpha_0 h(x) - \alpha_1 L_f h(x) - \alpha_2 L_f^2 h(x) + \mu(t)}{L_g L_f^2 h(x)}$$

Robust Sliding Mode Control



- Rejection of disturbances in the neighborhood of a desired operating point by means of sliding mode control regarding physical actuator constraints
- Online application of interval analysis to account for uncertainties in measurements and for state reconstruction errors
- Quality criterion for choosing adequate values for \dot{m}_{CG} and $\Delta \vartheta$ to manipulate the enthalpy flow of the cathode gas
- Online subdivision strategy allows for converting the interval-based controller output $[v(t)]$ into a point-valued system input $u(t)$
- Guarantee of asymptotical stability inspite of system uncertainties

Robust Sliding Mode Control



- Mathematical model of the SOFC system in an input-affine description is extended by a bounded disturbance $d \in [d]$

$$\begin{bmatrix} \dot{z}_1 \\ \dot{z}_2 \\ \dot{z}_3 \end{bmatrix} = \begin{bmatrix} z_2 \\ z_3 \\ \tilde{a}(z, p, d) \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \tilde{b}(z, p) \end{bmatrix} v$$

- Disturbance influences the system according to $\tilde{a} = L_f^3 h + d$
- Identification of interval parameters $p \in [p]$

Robust Sliding Mode Control

- Definition of an asymptotically stable sliding surface $s(\tilde{z}) = 0$ with the tracking error $\tilde{z}_1^{(j)} = z_1^{(j)} - z_{1,d}^{(j)}$

$$s(\tilde{z}) = \tilde{z}_1^{(2)} + \alpha_1 \tilde{z}_1^{(1)} + \alpha_0 \tilde{z}_1^{(0)} = 0$$

and the output z_1 and its time derivatives $z_1^{(j)}$, $j = 1, \dots, \delta - 1 = n_x - 1$

- Stabilization of the motion towards the sliding surface by a suitable Lyapunov function V

$$V = \frac{1}{2}s^2 > 0 \quad \text{for } s \neq 0, \quad \text{and its time derivative } \dot{V} = s\dot{s}$$

- The condition $\dot{V} = s\dot{s} \leq 0$ for the time derivative of the Lyapunov function is fulfilled with

$$s\dot{s} \leq -\eta s \operatorname{sign}\{s\} \quad \text{which is guaranteed for}$$

$$\dot{s} + \eta \cdot \operatorname{sign}\{s\} = -\beta \cdot \operatorname{sign}\{s\}, \quad \eta, \beta > 0$$

- Control input v is obtained from

$$\tilde{a}(z, p, d) + \tilde{b}(z, p)v - z_{1,d}^{(3)} + \alpha_1 \tilde{z}_1^{(2)} + \alpha_0 \tilde{z}_1^{(1)} = -(\beta + \eta) \cdot \operatorname{sign}\{s\}$$

Robust Sliding Mode Control

- Control law for the disturbance rejection in the thermal subsystem

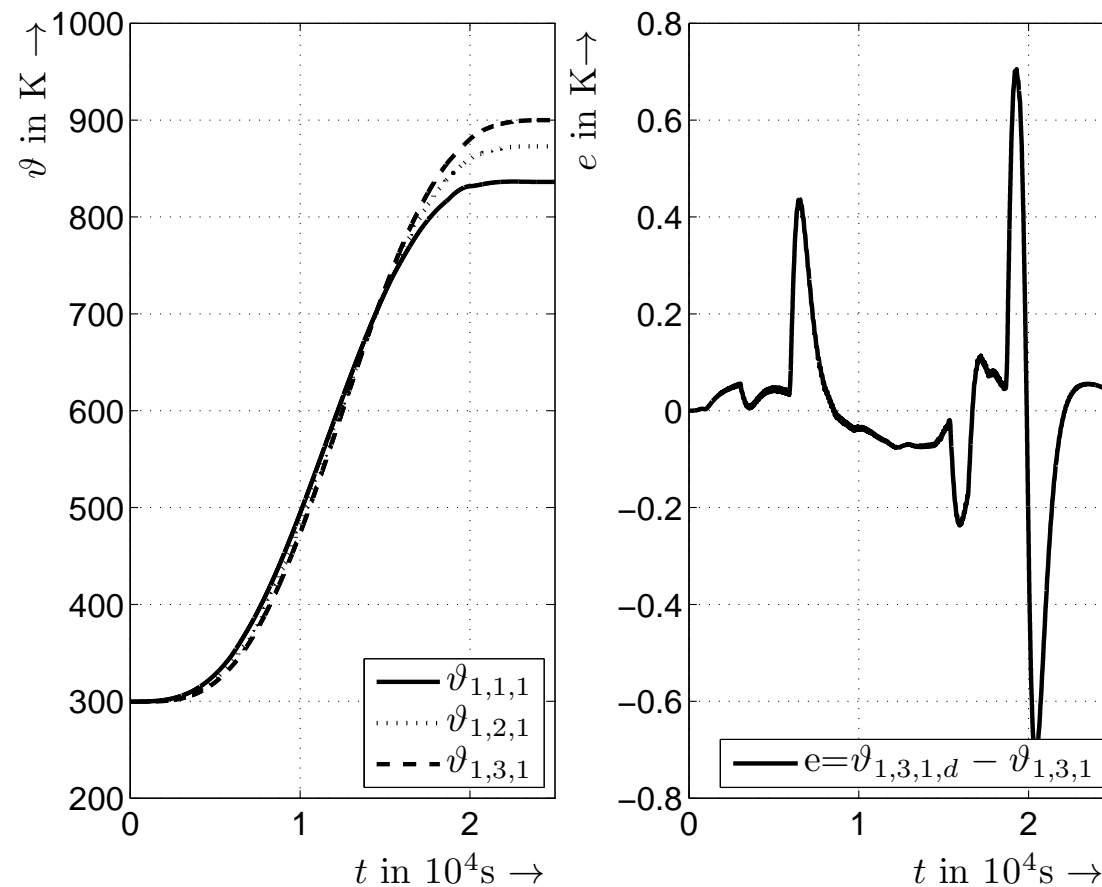
$$[v] := \left[\frac{-\tilde{a}(z, [p], [d]) + z_{1,d}^{(3)} - \alpha_1 \tilde{z}_1^{(2)} - \alpha_0 \tilde{z}_1^{(1)}}{\tilde{b}(z, [p])} - \frac{1}{\tilde{b}(z, [p])} \underbrace{(\eta + \beta)}_{=:\tilde{\eta} > 0} \cdot \text{sign}\{s\} \right] \Bigg|_{\substack{p \in [p] \\ d \in [d]}}$$

- Appropriate choice of the switching amplitude $\tilde{\eta}$ in the case of control design for interval parameters $p \in [p]$ and interval disturbance $d \in [d]$
- Controller output for a guaranteed stabilization of the thermal SOFC system

$$v := \begin{cases} \sup\{[v]\} & \text{for } s \geq 0 \\ \inf\{[v]\} & \text{for } s < 0 \end{cases}$$

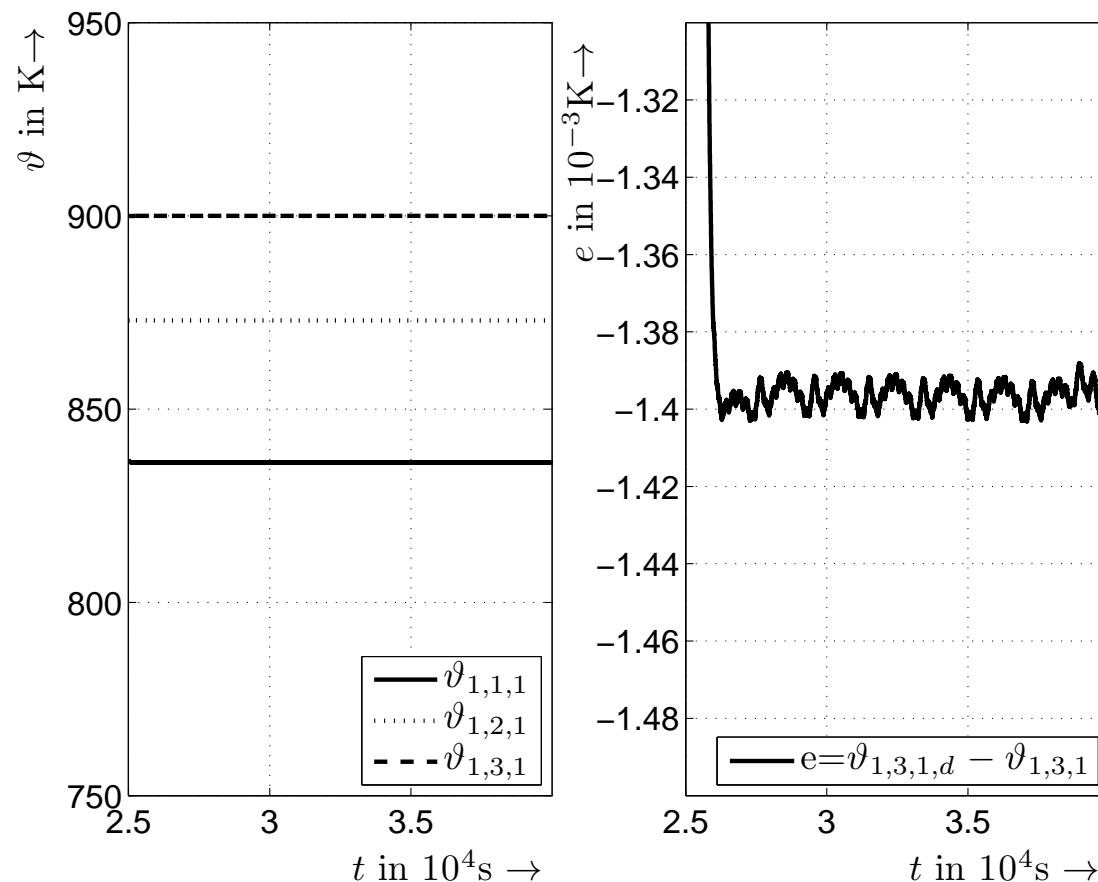
Robust Sliding Mode Control

- Instationary heating phase of the SOFC stack module using an exact linearizing control law to reach a desired operating point



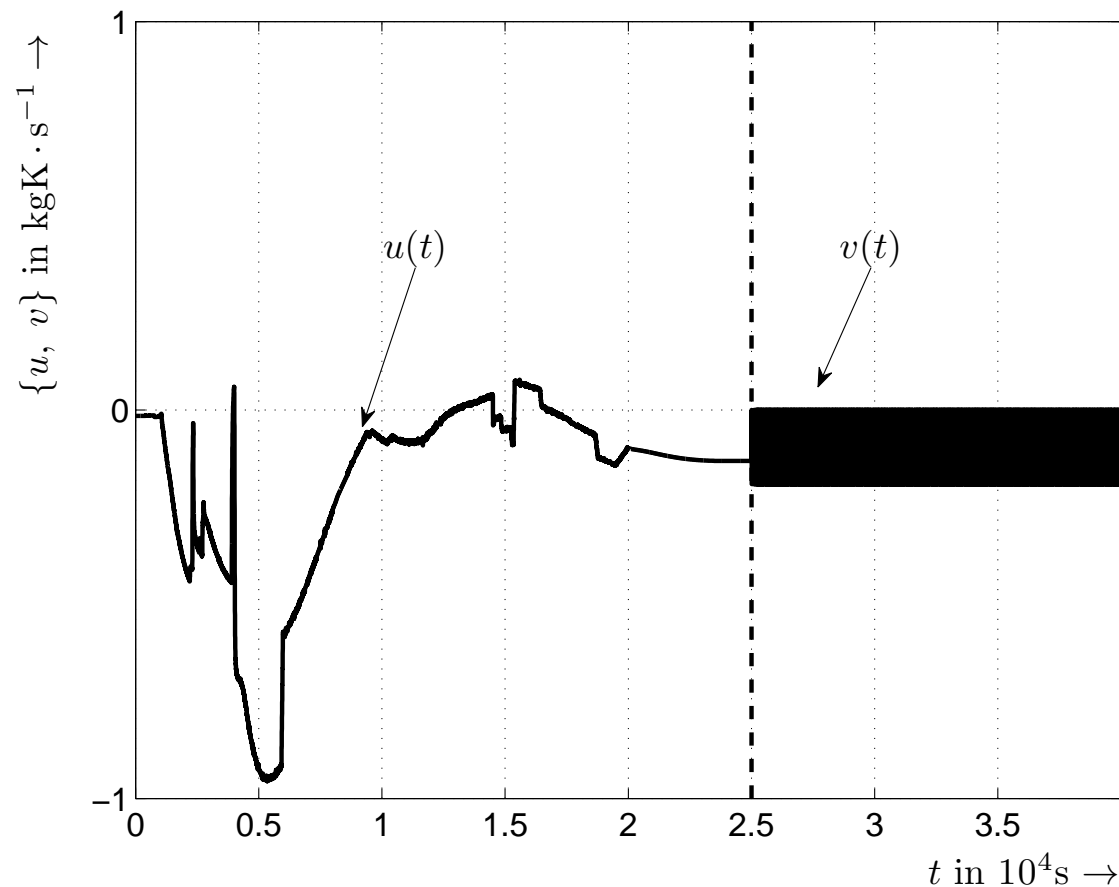
Robust Sliding Mode Control

- Switching to the interval-based sliding mode control law in the point of time $t = 2.5 \cdot 10^4$ s
- **Objective:** Rejection of disturbances and stabilization of desired operating points accounting for parameter uncertainties and bounded state enclosures



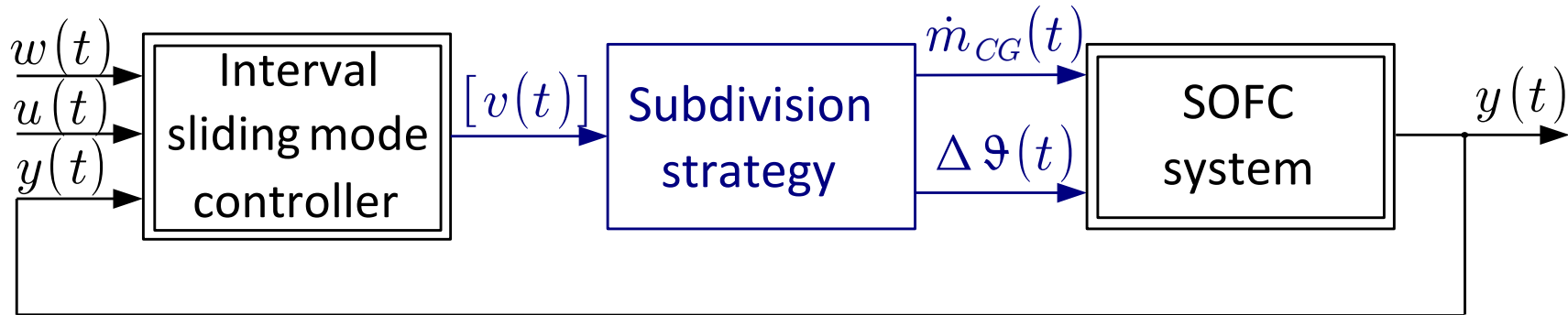
Robust Sliding Mode Control

- Output of exact linearizing feedback control law $u(t)$ with switching to the output of the interval-based sliding mode controller $v(t)$ at the point of time $t = 2.5 \cdot 10^4 \text{s}$



Problem: Adequate setting of the SOFC system input $u = \dot{m}_{CG} \Delta \vartheta$ with an available sliding mode controller output $v(t)$

Robust Sliding Mode Control



- Subdivision strategy to determine appropriate control inputs \dot{m}_{CG} and $\Delta \vartheta$ corresponding to $[v(t)]$
- The product of the mass flow \dot{m}_{CG} and of the temperature difference $\Delta \vartheta$ determines the system input

$$u := (\dot{m}_{CG} \cdot \Delta \vartheta)$$

- Operating ranges of the actuators are defined by bounded intervals

Implementation of the Interval-Based Control Law in Simulations

- A **splitting procedure** is employed in each time step k starting with the initial interval box described by $[\dot{m}_{CG}^{<0>}]$ and $[\Delta\vartheta^{<0>}]$ which is identical to the physical actuator constraints
- Multi-sectioning of the input interval vector $\left[[\dot{m}_{CG}^{<l>}] ; [\Delta\vartheta^{<l>}] \right]^T$ into the four interval boxes the mass flow and temperature difference in the time step k

$$\left[[\dot{m}_{CG}^{<l>}] ; [\Delta\vartheta^{<l>}] \right]^T := \begin{bmatrix} [\inf([\dot{m}_{CG}^{<l>}]) ; \text{mid}([\dot{m}_{CG}^{<l>}])] \\ [\inf([\Delta\vartheta^{<l>}]) ; \text{mid}([\Delta\vartheta^{<l>}])] \end{bmatrix}$$

$$\left[[\dot{m}_{CG}^{<L+1>}] ; [\Delta\vartheta^{<L+1>}] \right]^T := \begin{bmatrix} [\text{mid}([\dot{m}_{CG}^{<l>}]) ; \sup([\dot{m}_{CG}^{<l>}])] \\ [\inf([\Delta\vartheta^{<l>}]) ; \text{mid}([\Delta\vartheta^{<l>}])] \end{bmatrix}$$

$$\left[[\dot{m}_{CG}^{<L+2>}] ; [\Delta\vartheta^{<L+2>}] \right]^T := \begin{bmatrix} [\inf([\dot{m}_{CG}^{<l>}]) ; \text{mid}([\dot{m}_{CG}^{<l>}])] \\ [\text{mid}([\Delta\vartheta^{<l>}]) ; \sup([\Delta\vartheta^{<l>}])] \end{bmatrix}$$

$$\left[[\dot{m}_{CG}^{<L+3>}] ; [\Delta\vartheta^{<L+3>}] \right]^T := \begin{bmatrix} [\text{mid}([\dot{m}_{CG}^{<l>}]) ; \sup([\dot{m}_{CG}^{<l>}])] \\ [\text{mid}([\Delta\vartheta^{<l>}]) ; \sup([\Delta\vartheta^{<l>}])] \end{bmatrix}$$

Implementation of the Interval-Based Control Law in Simulations

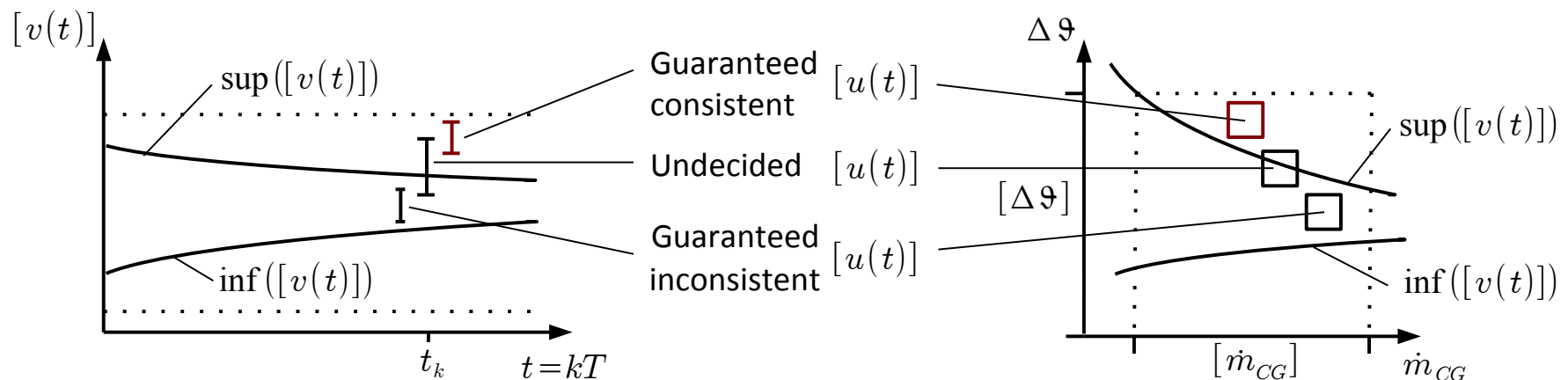
- A **validity test** of $[u^{<l>}] = [\dot{m}_{CG}^{<l>}] [\Delta\vartheta^{<l>}]$ is performed according to the controller output $[v]$ for classifying **guaranteed consistent**, **undecided** and **guaranteed inconsistent** input intervals

- Consistency of $[u^{<l>}]$ in $[v]$ is proven if

$$\sup\{[v]\} < \inf\{[u^{<l>}]\} \quad \text{for } s \geq 0$$

$$\inf\{[v]\} > \sup\{[u^{<l>}]\} \quad \text{for } s < 0$$

- Illustration of the consistency test for $s > 0$ bounded by actuator constraints (dashed lines)



Implementation of the Interval-Based Control Law in Simulations

- Compositions of $u(t)$ are assessed for l subdivided intervals in each time step k
- Optimal interval box of $[\dot{m}_{CG}]$ and $[\Delta\vartheta]$ is detected with the quality criterion

$$[J_k^{<l>}] = \kappa_1 \left([\Delta\vartheta_k^{<l>}] - [\Delta\vartheta_{nom}] \right)^2 + \kappa_2 \left([\Delta\vartheta_k^{<l>}] \right)^2 + \kappa_3 \left([\dot{m}_{CG,k}^{<l>}] - [\dot{m}_{nom}] \right)^2$$

- The minimization of $J_{opt} = \min \left(\inf \left([J_k^{<l>}] \right) \right)$ yields

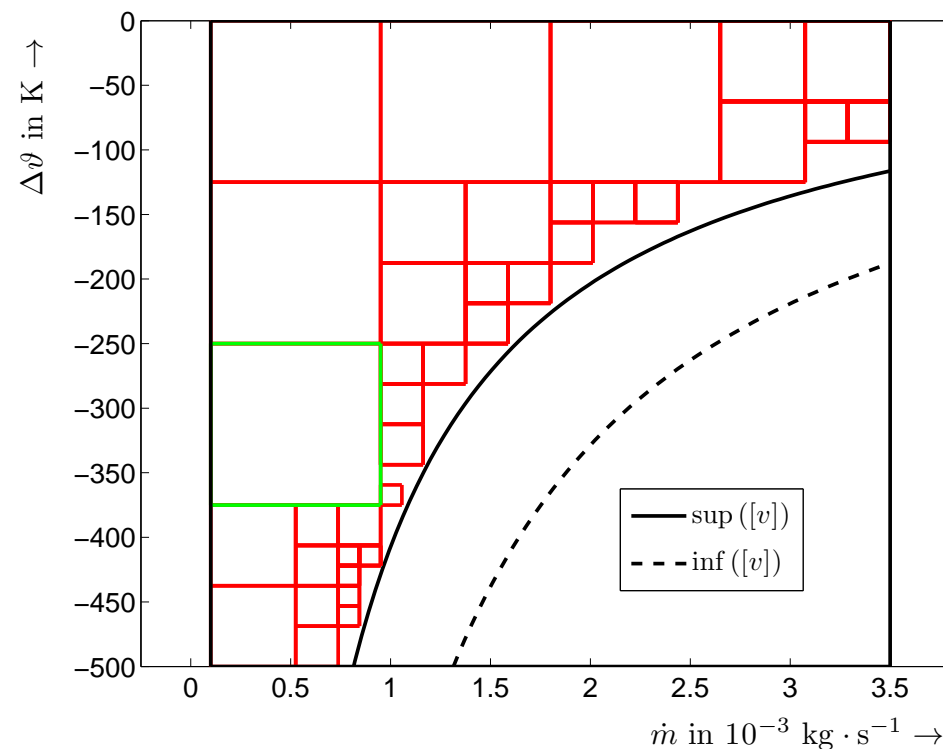
$$[\dot{m}_{CG}^{<opt>}] \quad \text{and} \quad [\Delta\vartheta^{<opt>}]$$

- The guaranteed stabilizing control output for the SOFC system with $v \geq \sup([v])$ is determined by

$$u(t) = \text{mid} \left([\dot{m}_{CG}^{<opt>}] \cdot [\Delta\vartheta^{<opt>}] \right)$$

Implementation of the Interval-Based Control Law in Simulations

- Depiction of the optimal system input with reference to the nominal values for $[\dot{m}_{nom}]$ and $[\Delta\vartheta_{nom}]$
- Cooling process with a value $s > 0$ in the sliding mode control design



Conclusions and Outlook

Conclusion

- Nonlinear modeling of the thermal subsystem of SOFCs including uncertainties in the parameterization and the system states
- Design of an interval-based sliding mode controller that is capable to cope with bounded uncertainties in a desired operating point
- Optimal adjustment of the enthalpy flow as a control input of the system employing a subdivision strategy regarding actuator constraints

Dötschel, Thomas; Rauh, Andreas; Aschemann, Harald: *Reliable Control and Disturbance Rejection for the Thermal Behavior of Solid Oxide Fuel Cell Systems*, presented at MATHMOD 2012, Vienna, Austria, 2012. to appear on IFAC-PapersOnLine.net

Conclusions and Outlook

Outlook

- Proof of the robustness in case of a switching output y where the remaining system dynamics have to be enclosed in state intervals
- Implementation of the presented approaches in the SOFC system available at the Chair of Mechatronics at the University of Rostock
- Translation of the software routines in INTLAB into the real-time capable C-XSC implementation

Rauh, Andreas; Aschemann, Harald: *Interval-Based Sliding Mode Control and State Estimation for Uncertain Systems*, IEEE Intl. Conference on Methods and Models in Automation and Robotics MMAR 2012, Miedzyzdroje, Poland, 2012.